Anti-Windup Algorithms for Pilot-Induced-Oscillation Alleviation

Isabelle Queinnec, Sophie Tarbouriech, Jean-Marc Biannic, Christophe Prieur

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The paper deals with the development of anti-windup schemes and related numerical oriented tools. The objective is then to design anti-windup compensators to guarantee stability and performance for some particular classes of nonlinear actuators presenting both magnitude and rate saturations. The lateral flying case for a civil aircraft undergoing aggressive maneuvering by the pilot is addressed. A complete methodology including theoretical conditions and associated toolbox is then proposed and compared to solutions based on anti-PIO filters.

Introduction

The paper is aimed at developing anti-windup schemes and related numerical tools, in order to alleviate the Pilot-Induced-Oscillations in the lateral flying case for a civil aircraft undergoing aggressive maneuvering by the pilot. Indeed, anti-windup strategies represent an appropriate framework to mitigate the undesired saturation effects [24], [31]. Thus, the general principle of the anti-windup scheme can be depicted in Figure 1, where the (unconstrained) signal produced by the controller is compared to what is actually fed into the plant (the constrained signal). This difference is then used to adjust the control strategy by preserving stability and performance.

Actually, this problem is particularly crucial for the space and aeronautical fields, where the Pilot-Induced-Oscillation (PIO) phenomenon is observed; that is, the existence of a particular external excitation (signal $w$ in the notation of Figure 1) renders the closed-loop system unstable without anti-windup compensator [1], [15]. Thus, ad-hoc [22], [16] or advanced anti-windup strategies [8], [9], [18] for PIO suppression have been proposed in the literature and applied in practice.

For a given plant in closed loop with a pre-designed controller (designed without taking into account the saturation constraint) and a saturating input, the design of the anti-windup compensator is usually split into two steps, as explained in [26]. First, an analysis study is performed to estimate the effect of the isolated nonlinearity on the performance of the closed-loop system. Then, the second step is the design of an anti-windup compensator to improve the performance. By "performance", various notions could be considered, such as the $L_2$ gain between a perturbation $w$ and the regulated output $z$, as depicted in Figure 1. Of course, this performance estimation is associated with an estimation of the region where the initial conditions need to be restricted, in order to guarantee the asymptotic or exponential stability of the origin.

Numerous methods exist for the design of anti-windup compensators for control systems in the presence of magnitude or rate saturation constraints. See, for example, [11], [30], [10], [28], [24], [31] to cite just a few books, surveys or special issues on this subject. Of course, the aim of this paper is not to give an exhaustive perspective about anti-windup compensator design, but rather to present some hints and algorithms on how to solve numerically the anti-windup compensator design problem for an application purpose. Actually, due to the classical tradeoff between performance and estimation of the suitable region of initial conditions, the design of anti-windup...
compensators is cast into a static optimization problem, written in terms of Linear Matrix Inequalities (LMIs). Such an optimization problem can be solved numerically in an efficient way using classical software in a Matlab environment. To illustrate the approach and algorithms, the anti-windup compensator design methods are applied to a lateral aircraft model, in order to provide a systematic way to mitigate the PIO phenomenon. Although actuator loss is not exactly the subject of the paper, we illustrate the case where only one actuator is available, allowing us to consider a harsh limit on the actuator bounds to better exhibit the effect of saturation and anti-windup actions.

This paper is organized as follows. First, the model and the problem under consideration are stated in Section "Model description and problem formulation". The main results are presented in Section "Main anti-windup design conditions", where numerically tractable conditions are given to solve the anti-windup compensator design problem and some efficient algorithms are given. The numerical tools used to actually solve the problem are focused on in Section "Dedicated software tools for solving saturated and anti-windup problems". These tools are then illustrated through an application to a realistic model for a civil transport aircraft in Section "PIO alleviation using an anti-windup loop". Some concluding remarks and perspectives end the paper.

Model description and problem formulation

The full model, including the plant, actuator, controller and anti-windup loop, is precisely defined below.

Plant model

We assume that the output of the controller is not affected in a same way by the nonlinear elements. The vector \( u \in \mathbb{R}^m \) building the \( m \) inputs of the plant is broken down into two subvectors: the first one, denoted by \( u_\text{s} \in \mathbb{R}^{n_\text{s}} \), corresponds to \( n_\text{s} \) saturated inputs, whereas the second one, denoted by \( u_\text{n} \in \mathbb{R}^{n_\text{n}} \), corresponds to the linear inputs (unsaturated inputs). The plant model can be defined by:

\[
\begin{align*}
\text{sysP} : & \quad \dot{x}_p = A_p x_p + B_p u_s + B_p^w w + B_p^v v, \\
& \quad y_p = C_p x_p + D_p u_s + D_p^w w + D_p^v v, \\
& \quad z = C z_p + D u_s + D^w w + D^v v
\end{align*}
\]

where \( x_p \in \mathbb{R}^{n_p} \) and \( w \in \mathbb{R}^1 \) are the state and the measured output of the plant. \( w \in \mathbb{R}^1 \) generally represents an exogenous perturbation, but may also be used to represent a reference signal (or both). Furthermore, \( z \in \mathbb{R}^1 \) represents the regulated output, which is used to evaluate the performance of the system with respect to the perturbation \( w \) via some appropriate optimization criteria.

Controller model

Unlike the classical anti-windup loops, in which the output of the anti-windup controller is injected into the dynamics of the controller and/or the output of the controller, we consider here that the output of the anti-windup controller modifies only partially the dynamics of the controller and/or the output of the controller. Thus, the dynamical controller is described as follows:

\[
\begin{align*}
\text{sysC} : & \quad \dot{x}_c = A_c x_c + B_c u_c + B_c^w w + B_c^v v, \\
& \quad y_c = C_c x_c + D_c^w u_c + D_c^w w + D_c^v v, \\
& \quad y_{cs} = C_c x_c + D_c^w u_c + D_c^w w
\end{align*}
\]

where \( x_c \in \mathbb{R}^{n_c} \) and \( u_c \in \mathbb{R}^m \) are the state and the input of the controller. The output of the controller is broken down into two signals: \( y_{cs} \in \mathbb{R}^{n_c} \), which will be interconnected \( u_s \) through a saturated actuator, and \( y_{cs} \in \mathbb{R}^{n_c} \), which will be interconnected with the linear (unsaturated) input \( u_n \). Moreover, \( v \) and \( v_s \) are the additional inputs that will be connected to the anti-windup controller. \( B_a \) and \( D_a \) are matrices of dimensions \( n_c \times n_c \) and \( m_c \times m_c \), and make it possible to specify what the \( n_c \) states and \( m_c \) outputs modified by the anti-windup action are.

Actuator model

The actuator block between the output of the controller \( y \), and the input of the plant \( u \) is divided into two blocks: the first one corresponding to the nonlinear (saturated) part and the second one corresponding to the linear (unsaturated) part. The nonlinear actuator part involves \( n_{cr} \)-nested saturations, including the case of rate and magnitude saturations, as depicted in Figure 2(a). Such nonlinearities are tackled via the use of dead-zone, denoted by \( \varphi (\cdot) \), \( i = 1, \ldots, n_{cr} \).

\[
\begin{align*}
& \quad \phi (\cdot) = (\phi_1 (\cdot), \ldots, \phi_{n_{cr}} (\cdot)) \quad \text{and} \quad \varphi (\cdot) = (\varphi_1 (\cdot), \ldots, \varphi_{n_{cr}} (\cdot)) \\
& \quad \phi_i (\cdot) = \begin{cases} 
0 & \text{if } |v_i| < 1 \\
\text{sat}_r (v_i) & \text{if } 1 < |v_i| \leq 2 \\
\text{sat}_m (v_i) & \text{if } 2 < |v_i|
\end{cases} \quad \text{and} \quad \varphi_i (\cdot) = \begin{cases} 
0 & \text{if } |v_i| < 1 \\
\text{sat}_r (v_i) - \text{sat}_m (v_i) & \text{if } 1 < |v_i| \leq 2 \\
\text{sat}_m (v_i) & \text{if } 2 < |v_i|
\end{cases}
\end{align*}
\]

The dynamical model of the actuator is based on Scheme 2(b) as follows:

\[
\begin{align*}
\text{sysACT} : & \quad \dot{v} = v + \varphi (v) \\
& \quad u_s = x_a
\end{align*}
\]

with

\[
\begin{align*}
& \quad \varphi (v) = T_0 R_{sat} (v) - T_0 R_{sat} (v) \\
& \quad \text{where } R_{sat} (v) = \begin{cases} 
0 & \text{if } |v| < 1 \\
\text{sat}_r (v) & \text{if } 1 < |v| \leq 2 \\
\text{sat}_m (v) & \text{if } 2 < |v|
\end{cases}
\end{align*}
\]

Anti-windup compensator

In the DLAW (Direct Linear Anti-Windup) strategy, the anti-windup controller uses as input the difference between the signals issued either from the input and the output of the whole actuator or from the input and the output of the nonlinear elements included in the actuator. Then, the anti-windup loop under consideration in the paper considers that the inputs of the anti-windup controller are the dead-zones associated with each saturation. Hence, the anti-windup controller of order \( n_{cr} \) is written as:

\[
\begin{align*}
& \quad A_{aw} x_{aw} + A_{aw}^v \varphi (v) + A_{aw}^v \varphi (v) \\
& \quad v \quad y = A_{aw} x_{aw} + A_{aw}^v \varphi (v) + A_{aw}^v \varphi (v)
\end{align*}
\]

where \( v_s \) and \( v_c \) are of dimensions \( n_{cr} \) and \( m_{cr} \), respectively.

Interconnections

The interconnections considered can be described as follows:

- linear link between the output of the plant and the input of the controller: \( u_s = y_p \).
the first part of the output of the controller ($y_c$) is linked to the corresponding inputs of the plant ($u_i$) through the actuator model (3);
- the second part of the output of the controller is directly connected to the corresponding inputs of the plant: $u_w = y_{ext}$;
- $\nu_c$ and $\nu_w$ are built from the anti-windup compensator.

**Remark 2.1**
An important fact is that the anti-windup model (4) imposes the assumption that the input and output signals of each saturation block in Figure 2 are available. To overcome this assumption, alternative strategies can be investigated. For example, the anti-windup may use the difference between the nonlinear actuator and a linear fictitious one (with the same dynamics but without saturation blocks) to explicitly take into account the dynamics of the actuator (present in the rate limiter) [20]. Another option would be to build an observer to evaluate the internal state of the actuator [27].

**Standard formulation**

In [24], a standard formulation of the anti-windup design has been proposed for various kinds of actuators. In the current case, by considering an augmented state of dimensions $n = n_u + m + n_w + n_{aw}$ including the state of the plant, the state of the actuator, the state of the controller and the state of the anti-windup controller, the following standard model of the complete closed-loop system can be defined by:

$$
\begin{aligned}
\dot{x} &= \mathcal{A}x + \mathcal{B}_x(y_c + y_v) + \mathcal{B}_w w \\
y_c &= \mathcal{C}_x x + D_{x0} y_c + D_{x1} y_v + D_{xw} w \\
y &= \mathcal{C}_y x + D_{y0} y_c + D_{y1} y_v + D_{yw} w \\
z &= \mathcal{C}_z x + D_{z0} y_c + D_{z1} y_v + D_{zw} w
\end{aligned}
$$

(5)

where the matrices of the anti-windup controller are encapsulated into the matrices of system (5). Details of these matrices are given in Section “Algorithms for $AW_c$ design”.

The design procedure of the anti-windup controller consists in optimizing some quantities, such as the size of the region of stability of the closed-loop system or the guaranteed level of performance. In particular, the idea when adding the anti-windup loop is to maximize the basin of attraction of the origin for the closed-loop system and/or to minimize the $L_2$ gain between $w$ and $z$ or to maximize the set of perturbation $w$, which can be rejected. Then, the perturbation signal is assumed to be bounded in energy, as follows:

$$
\|w\|_2^2 = \int_0^T w(t)^T w(t) dt \leq \delta^{-1}; \quad 0 \leq \delta^{-1} < \infty
$$

(6)

The problem that we intend to address is summarized below.

**Problem 2.2**

Determine an anti-windup controller $AW_i$ and a region $\mathcal{E}$, as large as possible, such that

- **Internal stability.** When $w = 0$, the closed-loop system (5) is asymptotically stable for any initial conditions belonging to $\mathcal{E}$ (which is a region of asymptotic stability (RAS));
- **Performance.** When $w \neq 0$, satisfying (6), and for $x(0) = 0$, the $L_2$ gain between $w$ and $z$ is finite and equal to $\gamma > 0$. Furthermore, the trajectories of the closed-loop system (5) remain bounded in the set $\mathcal{E}$.

The convex optimization problems associated with Problem 2.2 are specified in Section “Algorithms for $AW_i$ design”.

**Main anti-windup design conditions**

**Solution to standard anti-windup design**

The following proposition provides conditions of local stability and $L_2$ performance for the closed-loop system (5). The result considers existence conditions to solve Problem 2.2.

**Proposition 3.1**

If there exist a symmetric positive definite matrix $Q \in \mathbb{R}^{n \times n}$, two matrices $Z_i$ and $Z_i$ such that $Q_{ij}^T - Z_i^T Z_i$, and $S_i \in \mathbb{R}^{m \times m}$ and $S_i \in \mathbb{R}^{m \times m}$ and a positive scalar $\gamma$ such that the following conditions are met:

$$
\begin{bmatrix}
Q A^T + AQ & B_S S_0 - QC_0 - Z_0 & B_S S_C - Z_0 & B_S Q C_2^T \\
-2S_0 - D_{s0} S_0 - S_{d0} & -D_{s1} S_0 - S_{d1} & -D_{s2} S_{d2}^T & \\
-2S_1 - D_{s1} S_1 - S_{d1} & -D_{s2} S_{d2}^T & -D_{s2} S_{d2}^T & < 0
\end{bmatrix}
$$

(7)

then,

1. when $w = 0$, the set $\mathcal{E} = \{ x \in \mathbb{R}^n; x^T Q x \leq \delta^{-1} \}$ is RAS for the closed-loop system (5);
2. when $x(0) = 0$, satisfying (6), and for $x(0) = 0$,
   - the trajectories of the closed-loop system remain bounded in the set $\mathcal{E}$;
   - the $L_2$ gain is finite and one obtains:

$$
\int_0^T z(t)^T z(t) dt \leq \gamma \int_0^T w(t)^T w(t) dt, \quad \forall T \geq 0
$$

(10)

The detailed way to derive the conditions and to prove them can be found, for example, in [24], [31].

**Remark 3.2**

The interest of the anti-windup structure resides in the simplicity of the design conditions. Indeed, the design of a static anti-windup gain (only matrices $D_{s0}$ and $D_{s1}$ are used) is the result of a fully linear problem. In the case of the design of a dynamical anti-windup controller, for a priori given matrices $A_{aw}$ and $C_{aw}$, the determination of the input and transmission matrices is also obtained by solving a linear problem. In the case where $n_{aw} = n_u + n_w + n_z$, the resolution of a linear problem can also be considered through an iterative procedure [24].

For analysis purposes (the anti-windup controller being given), the conditions of Proposition 3.1 are linear and can be directly used to solve adequate optimization problems. Moreover, in the design context, the conditions of Proposition 3.1 are non-convex, matrices $A_{aw}$, $B_{aw}$, $C_{aw}$ and $D_{aw}$, hidden in matrices $A$, $B$, $C$, $D$, $i$, $j = 0, 1$. Conditions with linear decision variables can be obtained, more or less directly, by slightly modifying the original conditions, or even by considering iterative procedures (including D-K iteration process).
allowing a Lyapunov matrix and anti-windup matrices to be sought. These situations are detailed in the next subsection.

Remark 3.3
In the sequel, one considers a set $\mathcal{X}_0$, defined by some directions in the plant state space $v_i \in \mathbb{R}^n$, $i = 1,...,q$, to provide a desired shape of the region $E(Q^\top, \delta)$ to be maximized when solving Problem 2.2.
Then, considering $v_i = [v_i^T, 0]^T \in \mathbb{R}^n$, $i = 1,...,q$ and $\beta$ a scaling factor such that $\beta v_i \in E(Q^\top, \delta)$, $i = 1,...,q$ (which corresponds to imposing $\beta \mathcal{X}_0 \subset E(Q^\top, \delta)$), an additional condition to those of Proposition 3.1 must be considered in the algorithms, as follows:

$$\begin{bmatrix}
\delta_1 \\
\delta_{v_i} \\
\delta_{v_i}^T \\
Q
\end{bmatrix} > 0, \ i = 1,...,q$$

This means that $\beta$ is used to maximize the region of attraction of the system and $\mathcal{X}_0$ allows the directions of interest for this region of attraction to be oriented.

Algorithms for $AW_i$ design

From (1), (2), (3) and (4), the matrices of system (5) are defined by:

$$A = \begin{bmatrix}
\mathbb{A} & B C_{aw} \\
0 & A_{aw}
\end{bmatrix}, \quad B_i = \begin{bmatrix}
B_{pi} + B D_{aw} \\
B_{aw}^0
\end{bmatrix}, \quad B = \begin{bmatrix}
B_{pi} + B D_{aw}^1 \\
B_{aw}^1
\end{bmatrix}$$

$$C_i = \begin{bmatrix}
C_0 & C_{aw} C_{aw}
\end{bmatrix}, \quad C_i = \begin{bmatrix}
C_i & C_{aw} C_{aw}
\end{bmatrix}, \quad C_2 = \begin{bmatrix}
C_2 & 0
\end{bmatrix}$$

$$D_{aw} = C_{aw} D_{aw}^0, \quad D_{aw} = D_{aw}^0, \quad D_{aw} = D_{aw}^0, \quad D_{aw} = D_{aw}^0$$

$$B_2 = \begin{bmatrix}
0 \\
0
\end{bmatrix}, \quad D_2 = 0, \quad D_2 = 0$$

with

$$A = \begin{bmatrix}
A_p + B_{pw} D_{aw}^{-1} D_{aw}^{\top} C_p \\
T_{aw} D_{aw} (C_p + D_{aw}^{-1} D_{aw}^{\top} C_p) \\
B C_p + B_{pw} D_{aw}^{-1} D_{aw}^{\top} C_p
\end{bmatrix}$$

$$\mathbb{A} = \begin{bmatrix}
A_p + B_{pw} D_{aw}^{-1} D_{aw}^{-1} D_{aw}^{\top} D_{aw} \\
T_{aw} (D_{aw} D_{aw}^{-1} D_{aw}^{\top} D_{aw} - I) + D_{aw} D_{aw}^{-1} D_{aw}^{\top} D_{aw} \\
B_{pw} D_{aw}^{-1} C_p
\end{bmatrix}$$

$$B_2 = \begin{bmatrix}
0 \\
0
\end{bmatrix}, \quad B_2 = \begin{bmatrix}
0 \\
0
\end{bmatrix}, \quad D_2 = I_{ns}$$

$$C_0 = \begin{bmatrix}
D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p) \\
T_{aw} D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p) \\
D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p)
\end{bmatrix}$$

$$C_1 = \begin{bmatrix}
D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p) \\
T_{aw} D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p) \\
D_{aw} (I_p + D_{aw}^{-1} D_{aw}^{\top} C_p) - T_{aw} T_0 (C_p + D_{aw}^{-1} D_{aw}^{\top} C_p)
\end{bmatrix}$$

$$C_2 = \begin{bmatrix}
D_{aw}^{-1} D_{aw}^{\top} C_p \\
D_{aw}^{-1} D_{aw}^{\top} C_p \\
D_{aw}^{-1} D_{aw}^{\top} C_p
\end{bmatrix}$$

$$D_{au} = D_{aw} + D_{aw}^{-1} D_{aw}^{\top} (D_{aw} D_{aw}^{\top} D_{aw}^{\top})$$

$$D_{aw} = D_{aw} + D_{aw}^{-1} D_{aw}^{\top} (D_{aw} D_{aw}^{\top} D_{aw}^{\top})$$

Furthermore, matrices defining the interconnection between the anti-windup loop and the system are:

$$B_r = \begin{bmatrix}
0 \\
T_{aw} D_{aw} \begin{bmatrix}
0 & I_{ns}
\end{bmatrix}
\end{bmatrix}, \quad C_{r0} = D_{ca} \begin{bmatrix}
0 & I_{ns}
\end{bmatrix}, \quad C_{r1} = T_{aw} D_{ca} \begin{bmatrix}
0 & I_{ns}
\end{bmatrix}$$
The analysis problem (Algorithm 3.4) is linear and the synthesis problem of the anti-windup is nonlinear, including products between decision variables and, in particular, between the Lyapunov matrix $Q$ and the anti-windup elements. A D-K iteration procedure may then be considered for the synthesis problem (Algorithm 3.6). However, the synthesis optimization problem may be partially linearized and, for given matrices $A_{aw}$ and $C_{aw}$, the design of matrices $B_{aw}$ and $D_{aw}$, $i = 0,1$ can be handled via a linear optimization problem (Algorithm 3.5).

Algorithm 3.4

Analysis of a given $AW_i$ anti-windup controller
1. Select matrices $A_{aw}$ and $C_{aw}$. A static anti-windup $AW_i$ may also be used by considering $n_{aw} = 0$.
2. Choose directions to be optimized $v_i \in \mathbb{R}^s$, $i = 1,\ldots,q$ and a known perturbation bound $\delta$.
3. Solve
   \[
   \min_{Q: S, X, S_0, X_0, Z, Z_0, B_{aw}, D_{aw}, \gamma, \mu} f_{cost}(\gamma, \mu) \\
   \text{subject to } LMI (7), (8), (9) \text{ and } (11)
   \]
   where $\gamma$ is the $L_2$ gain between $w$ and $z$ and $\mu = 1/\beta^2$.

Algorithm 3.5

Design of an $AW_i$ anti-windup controller with fixed dynamics
1. Select matrices $A_{aw}$ and $C_{aw}$. A static anti-windup $AW_i$ may also be used by considering $n_{aw} = 0$.
2. Choose directions to be optimized $v_i \in \mathbb{R}^s$, $i = 1,\ldots,q$ and a known perturbation bound $\delta$.
3. Solve
   \[
   \min_{Q: S, X, S_0, X_0, Z, Z_0, B_{aw}, D_{aw}, \gamma, \mu} f_{cost}(\gamma, \mu) \\
   \text{subject to } LMI (7), (8), (9) \text{ and } (11)
   \]
   where $\gamma$ is the $L_2$ gain between $w$ and $z$ and $\mu = 1/\beta^2$.
4. Compute $B_{aw} = B_{aw} S_0^{-1}$, $B_{aw} = B_{aw} S_0^{-1}$, $D_{aw} = D_{aw} S_0^{-1}$ and $D_{aw} = D_{aw} S_0^{-1}$.

Algorithm 3.6

Design of an $AW_i$ anti-windup controller – full design
1. Select matrices $A_{aw}$ and $C_{aw}$ of appropriate dimensions, in order to build the desired anti-windup loop.
2. Choose the directions to be optimized $v_i \in \mathbb{R}^s$, $i = 1,\ldots,q$ and a known perturbation bound $\delta$.
3. Pre-synthesis step – Solve
   \[
   \min_{Q: S, X, S_0, X_0, Z, Z_0, B_{aw}, D_{aw}, \gamma, \mu} f_{cost}(\gamma, \mu) \\
   \text{subject to } LMI (7), (8), (9) \text{ and } (11)
   \]
   where $\gamma$ is the $L_2$ gain between $w$ and $z$ and $\mu = 1/\beta^2$.
4. Compute $B_{aw} = B_{aw} S_0^{-1}$, $B_{aw} = B_{aw} S_0^{-1}$, $D_{aw} = D_{aw} S_0^{-1}$ and $D_{aw} = D_{aw} S_0^{-1}$.
5. If the solution obtained is satisfactory (some accuracy has to be fixed), or no longer improved compared to the previous steps, then STOP. Otherwise, go to the next iteration (the idea is to finish by a pre-synthesis step).
6. Synthesis step – Pick the solution $Q$ obtained at Step 3 and solve
   \[
   \min_{S, X, S_0, X_0, A_{aw}, C_{aw}, B_{aw}, D_{aw}, \gamma} f_{cost}(\gamma, \mu) \\
   \text{subject to } LMI (7), (8), (9) \text{ and } (11)
   \]
7. Go to Step 3.

Remark 3.7

The optimization cost function $f_{cost}$ is typically related to the performance of the disturbance rejection ($\min_{\gamma}$) and/or to the size of the domain of safe behavior in which the trajectories of the system may be initiated. In this paper, we consider inequalities (11) and $\min_{\mu}$ with $\mu = 1/\beta^2$ but any other criterion of the matrix $B_{aw}$ could be used.

Remark 3.8

In Algorithm 3.5 and in Step 3 of Algorithm 3.6, condition (7) is not directly applied. The products between $B_{aw}$ and $D_{aw}$ with the matrices $S_i$ are replaced by the change of variables $B_{aw} S_i$ and $D_{aw} S_i$, $i = 0,1$, which allows the problem to be linearized.

Remark 3.9

An interesting case is the static anti-windup one, for which matrices $A_{aw}$ and $C_{aw}$ are null matrices of appropriate dimensions. It implies that $B_{aw}$, $i = 0,1$, are also null matrices of appropriate dimensions and only matrices $D_{aw}$, $i = 0,1$, are computed in Algorithm 3.5.

Remark 3.10

Matrices $A_{aw}$ and $C_{aw}$ to be used in Algorithm 3.5 may be selected as the solution to a full-order ($n_{aw} = n_p + n_c + m_p$) anti-windup compensator design where the actuator is just a saturation in magnitude (see, for example, the conditions provided in [24]), i.e., via a linear optimization problem. Eventually, an order-reduction step may also be considered in order to select matrices $A_{aw}$ and $C_{aw}$ (see Example 8.5 in [24]). Other procedures developed in [31], such as the Model Recovery Anti-Windup (MRAW), could be used.

Dedicated software tools for solving saturated and anti-windup problems

For numerical evaluations, Semi-Definite Programming (SDP) solvers are easily available in a Matlab environment, either considering the MathWorks® package LMI Lab included in the Robust Control Toolbox™ or any freely available solvers. Similarly, in addition to the original parser of the LMI Lab package, one may prefer YALMIP format [17] to specify LMIs systems, convex optimization costs and associated solvers.

SATAW toolbox [19] has been developed to perform analysis and control design operations for linear systems interconnected with saturation elements. The toolbox manipulates a flexible description of the continuous-time system, controller and actuator using simple structure elements, as they are described in Section "Main anti-windup design conditions". For the saturation modeling, sector conditions are used. In this representation, the saturation term is replaced by a dead-zone nonlinearity. Hence, sector conditions, locally or globally valid, can be used to provide stability and stabilization conditions. The package then includes several functions for:

- state feedback or output feedback analysis, in the presence of position saturation and/or rate saturation;
- state feedback or output feedback design, in the presence of position saturation;
- static and dynamic anti-windup analysis, in the presence of position saturation;
- static anti-windup design, in the presence of position saturation.
Actually, the current published version of the toolbox does not allow the dynamic anti-windup design problem to be formally solved in the presence of position and rate saturation, but gives many elements to extend the functions to the case addressed in this paper.

Alternatively, the AWAST Tools [5], which were recently updated and integrated as a library (SAW Library) of the SMAC Toolbox [4], enable rather general anti-windup problems to be formalized and solved, following the practical framework proposed in [6].

PIO alleviation using an anti-windup loop

The design and analysis algorithms of Section “Main anti-windup design conditions” are now applied and compared in the realistic context of lateral maneuvers of a civil transport aircraft. Specific attention is devoted to aggressive pilot demands in conjunction with actuator loss.

To do this, the pilot’s activity is modeled as a static gain $K_{pil}$. For this application, a normal activity would correspond to $K_{pil} = 1$. Actually, in stressful situations, notably in case of actuator loss, a more aggressive pilot behavior is generally observed, resulting in much higher gains. Here, the gain is set to $K_{pil} = 2$.

Problem setup and objectives

A nonlinear closed-loop Simulink implementation of the anti-windup structure is depicted in Figure 3. The “aircraft” block is a linearized version (for fixed airspeed and altitude under cruise flight conditions) of the lateral dynamics of the system, including structural filters and delays, resulting in a state-space model of dimension 63. The controller block includes a dynamical controller of dimension 29. Its central objective is to provide good damping for the Dutch roll and to enable a safe control of the roll rate so that the bank angle $\phi$ is then easily controlled by the pilot with a simple gain $K_{pil}$. The state-space models $sysP$ and $sysC$ are then readily obtained from the Simulink diagrams of Figure 3, with the help of the Matlab `linmod` function. The plant corresponds to the “yellow box” depicting the aircraft system while the global controller (including pilot actions) is obtained by extraction of the 3 blue boxes. A standard balanced reduction technique is finally applied to obtain reasonably sized models. The reduced orders obtained, respectively $n_p = 8$ and $n_c = 20$, are compatible with the proposed algorithms.

The aircraft system involves 2 control inputs ($m = 2$): ailerons and rudder deflections. Note that only the aileron deflection actuator is assumed to saturate ($m_s = 1$) for the considered maneuvers. Moreover, 5 outputs ($p = 5$) are available for feedback ($v = [\beta, p, r, \phi, \dot{\phi}]$). The performance is evaluated via the tracking accuracy on the fourth bank-angle output $\phi$ (then, $l = 1$). The disturbance input of System (1) is used to express the perturbing effect of the saturation of the system input, that is, $B_{pu} = B_{pu}^* (q = 1)$.

In the $AW_\phi$ strategy, two signals (one for the magnitude limitation and one for the rate limitation) are used by the anti-windup device. Their generation is detailed in the Simulink implementation of Figure 4.

The anti-windup controller acts on the internal dynamics of the nominal lateral controller of the aircraft through two scalar signals $v_r$ and $v_u$, which respectively affect roll and sideslip angle dynamics ($v_r = [v_r, v_u]$, and $v_u = 0$, $n_r = 2$, $m_r = 0$). This means that matrix $B_{ua}$ appearing in Equation (2) is of dimension $n_r \times 2$.

The chosen strategy offers some flexibility, with the possibility of a direct anti-windup action at the controller output. However, no significant improvement has been observed with this additional feature, which has thus not been further considered in this application. This means that $D_{ua}$ appearing in Equation (2) is equal to 0.
The main objective of this application is to design and evaluate anti-windup compensators to improve the aircraft response to roll angle solicitations while limiting oscillations despite actuator loss [18]. During such maneuvers, a significant control activity is observed on the ailerons. This is why the effects of saturations are modeled and taken into account for these actuators in the diagram of Figure 3, while no saturation is introduced on the rudders. The effects of saturations become even more penalizing in case of a partial loss of control capability. Assume indeed that the aircraft is controlled by a pair of ailerons on each wing, but that only one is operational. In that case, the activity of the remaining actuators is doubled, as well as the risk of magnitude and rate saturations. Then, the magnitude and rate limits in the following are halved. We consider \( u_0 = L_m = 10 \text{ deg} \) (instead of 20 under normal conditions) and \( u_i = L_r = 20 \text{ deg} \) (instead of 40).

In the following, various cases studies are implemented and compared:

- Unsaturated – A non-saturated case where saturation elements are removed allows an ideal behavior of the closed-loop system to be exhibited;
- Saturated – A saturated case without compensation strategy is used as the nominal behavior of the closed-loop saturated system;
- Anti-PIO filter – The standard anti-PIO filter used in the industry consists in adding a dynamical block with pre-saturation before the output of the nonlinear block (denoted by \( e \)).

Let us first consider the design of a static \( AW \) anti-windup where only matrices \( D_{aw}^0 \), \( i = 0, 1 \) (see Equation (4)), have to be designed (\( n_m = 0 \)). The main advantage is that Condition (7) becomes linear and that the anti-windup block does not involve any additional dynamics. The optimization problem is solved by considering the bound on perturbation \( \delta = 0.1 \) and \( v_i = \pm [C_i(4;4)0] \), corresponding to the roll angle, as the direction to be optimized over the set \( E\{Q^{-1} \delta \} \). Algorithm 3.5, followed by Algorithm 3.4, provide the following optimal solution:

\[
\begin{align*}
\text{Static } AW \text{ design: } & \gamma = 1.18110; \beta = 0.7808 \\
\text{with the static anti-windup gains: } & \\
D_{aw}^0 = \begin{bmatrix}
-14.7887 & -0.0042 \\
8.6544 & \text{14.7887}
\end{bmatrix} & D_{aw}^1 = \begin{bmatrix}
0.0042 & 0.0392 \\
0 & 0
\end{bmatrix}
\end{align*}
\]

which shows that the anti-windup hardly uses the rate saturation information. Moreover, it is interesting to perform the same analysis for the saturated closed-loop system without anti-windup. The feasibility is also obtained and the solution is:

\[
\begin{align*}
\text{Analysis without anti-windup: } & \gamma = 1.8560; \beta = 0.7796 \\
\end{align*}
\]

The solution with the static \( AW \) anti-windup described through \( \gamma \) and \( \beta \) as performance indicators does not appear to be much better than the one without anti-windup: the anti-windup allows \( \gamma \) to be decreased and \( \beta \) to be increased, but only slightly. This does not reflect, however, the simulations described below, which show that the anti-windup action significantly improves the transient behavior of the roll angle, avoiding a large overshoot and degraded time evolution.
with respect to the saturated case. The meaning of this is that the considered optimization criterion, which does not explicitly include the time response performance, does not exactly fit the analysis or design of the anti-windup loop. Nevertheless, considering criteria on time response performance is a difficult task and the optimization criterion used here gives a reasonable trade-off between stability guarantee, performance and time response.

Figures 5 and 6 illustrate the time evolution of the closed-loop system to a roll solicitation of 40 deg at time \( t = 1 \) s followed by a step of –60 deg at time \( t = 15 \) s and a step of +60 deg at time \( t = 30 \) s. The responses are compared by considering the case with saturation and no compensation (saturated), a standard anti-PIO strategy (anti-PIO filter) and the static \( AW_s \) anti-windup strategy. It is important to underline that a simple static anti-windup strategy enables better performance than the standard anti-PIO case to be obtained, which adds dynamics to the system.

![Figure 5](image1.png)

**Figure 5** – roll solicitation of +40 deg at time \( t = 1 \) s followed by a step of –60 deg at time \( t = 15 \) s and a step of +60 deg at time \( t = 30 \) s: comparison of the performance output for the un-saturated, saturated (no compensation of the saturation), standard anti-PIO and static anti-windup \( AW_s \) simulations

![Figure 6](image2.png)

**Figure 6** – roll solicitation of +40 deg at time \( t = 1 \) s followed by a step of –60 deg at time \( t = 15 \) s and a step of +60 deg at time \( t = 30 \) s: comparison of the saturating input for the un-saturated, saturated (no compensation of the saturation), standard anti-PIO and static anti-windup \( AW_s \) simulations.

**Design of a dynamic anti-windup \( AW_d \)**

To go further, let us now consider the design of a dynamic anti-windup \( AW_d \). The difficulty in this case is to initialize the iterative procedure described in Algorithm 3.6, or to select matrices \( A_{aw} \) and \( C_{aw} \) used in Algorithm 3.5. As for the static case, the optimization problem is solved by considering the bound on perturbation \( \delta = 0.1 \) and \( \nu = [C_p(\delta)] \) as the direction to be optimized over the set \( \epsilon(Q^+,\delta) \).

Let us first consider a very simple structure to set matrices \( A_{aw} \) and \( C_{aw} \), namely a modal structure for \( A_{aw} \) allowing its dynamics to be set slightly faster than those of the closed-loop linear system:

\[
A_{aw} = \begin{bmatrix}
-100 & 50 \\
-50 & -100 
\end{bmatrix}, \quad C_{aw} = \begin{bmatrix}
-10 & 0 \\
0 & 10 \\
0 & 0 
\end{bmatrix}
\]

Algorithm 3.5 followed by Algorithm 3.4 provides the following optimal solution:

**Dynamic \( AW_d \) design 1**: \( \gamma = 1.7863; \ \beta = 0.8334 \)

with the anti-windup terms:

\[
B_{aw}^0 = \begin{bmatrix}
9461.4 \\
-7872.5
\end{bmatrix}, \quad B_{aw}^1 = \begin{bmatrix}
60.6 \\
-32.9
\end{bmatrix}, \quad D_{aw}^0 = \begin{bmatrix}
438.2 \\
1009 \\
0
\end{bmatrix}, \quad D_{aw}^1 = \begin{bmatrix}
3.5 \\
0
\end{bmatrix}
\]

Another option is to use matrices \( A_{aw} \) and \( C_{aw} \), which are the solution to another dynamic anti-windup scheme implemented on the same application. The idea is to circumvent the nonlinear problem of the dynamic anti-windup by pre-selecting matrices \( A_{aw} \) and \( C_{aw} \) obtained from other approaches, when they exist, with the expectation of obtaining better results than with a “random” selection as done above. In the current case, we consider the solution obtained with a structured \( H_\ell \) design method [18], and previously applied on the same numerical evaluation [7]. In that case, Algorithm 3.5 gives matrices \( B_{aw}^i \) and \( D_{aw}^i \) \( i = 0,1 \) (not provided here for readability reasons) and the following optimal solution:

**Dynamical \( AW_d \) design 2**: \( \gamma = 1.7395; \ \beta = 0.9147 \)
A roll solicitation of 40 deg at time $t = 1$ s followed by a step of −60 deg at time $t = 15$ s and a step of +60 deg at time $t = 30$ s is considered to compare the results. The time responses of the roll angle for the case without saturation, with the $\mathcal{H}_\infty AW_p$ anti-windup resulting from [7] and the designed dynamic $AW_p$ anti-windups are plotted in Figure 7. Similarly, the time evolutions of the control input $\delta_c$ in these cases are depicted in Figure 8.

One can observe that the level of performance of the very well-tuned $\mathcal{H}_\infty AW_p$ anti-windup is slightly degraded in comparison with the two cases of $AW_p$ design, but it remains acceptable and close to the ideal response that would be obtained if no saturation was present in the actuator block. One can also remark that the dynamic anti-windup design makes it possible to speed-up the rising time (less than 6 seconds) toward the set-point, with respect to the static anti-windup design (more than 6 seconds), even with very basic anti-windup dynamics ($AW_p$ design 1).

**Complementary illustrations**

The rate-saturation effectiveness is illustrated in Figures 9 and 10, where the signals $\nu$ and $x_s$ are plotted, respectively, for the case with and without anti-windup. One can check that the anti-windup action both reduces the number of rate-saturation events and the amplitude of the signal $\nu$ entering the saturation element (see Equation (3)).

Moreover, the stick response of the system, i.e., the output of the pilot gain block, is plotted in Figure 11 to illustrate the efficiency of the anti-windup design. In this case, with a moderately aggressive pilot ($K_{pil} = 2$), one can check that the pilot workload increases in

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**Figure 8** – Roll solicitation of +40 deg at time $t = 1$ s followed by a step of −60 deg at time $t = 15$ s and a step of +60 deg at time $t = 30$ s: comparison of the saturating input for the un-saturated, saturated (no compensation of the saturation), $\mathcal{H}_\infty AW_p$ anti-windup and designed anti-windup $AW_p$ cases

**Figure 9** – Roll solicitation of +40 deg at time $t = 1$ s followed by a step of −60 deg at time $t = 15$ s and a step of +60 deg at time $t = 30$ s: comparison of the signal $\nu$ used in Equation (3) for the saturated (no compensation of the saturation) and designed static and anti-windup cases

---

**Figure 10** – Roll solicitation of +40 deg at time $t = 1$ s followed by a step of −60 deg at time $t = 15$ s and a step of +60 deg at time $t = 30$ s: comparison of the signal $x_s$ used in Equation (3) for the saturated (no compensation of the saturation) and designed static and anti-windup cases

**Figure 11** – Roll solicitation of +40 deg at time $t = 1$ s followed by a step of −60 deg at time $t = 15$ s and a step of +60 deg at time $t = 30$ s: comparison of the pilot stick output for the un-saturated, saturated (no compensation of the saturation) and designed static and anti-windup cases
is set to 5 s to generate sufficient excitation in the aircraft modes of motion.

This illustrates the situation where strong excitation of the lateral aircraft modes of motion may result in the instability of the saturation response. Anti-PIO and anti-windup strategies allow stability to be preserved. Moreover, the dynamic anti-windup strategy enables a good tracking of the reference to be preserved, when the standard anti-PIO and the static anti-windup induce degraded responses with overshoot even if stable.

**Conclusion**

An anti-windup design strategy has been proposed for systems involving both magnitude and rate saturations, and taking into consideration that such saturations elements only affect some of the inputs. Such a situation is illustrated on a lateral flying model of a civil aircraft undergoing moderately aggressive maneuvering by the pilot. For this kind of systems, it is well known that magnitude and rate saturations of the aileron deflection actuator may lead to an undesirable behavior, often called Pilot-Induced-Oscillation (PIO). Anti-windup compensators have been designed through adequate convex optimization schemes, and a comparison with given dynamic anti-PIO filters already developed has also been provided. The numerical evaluation has made it possible to show, first, that a static strategy provides better results than classical anti-PIO filters basically used in industry. Moreover, the increase in the complexity of dynamic anti-windup compensators (both in terms of structure and computation) is compensated by the fact that they make it possible to recover behaviors very close to that which would be obtained if the actuators were linear. In any case, there is room for future work, such as the design of other anti-windup schemes, which may include the parameter-varying approach [21] or reset controllers [25].
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Isabelle Queinnec is currently CNRS researcher at LAAS-CNRS, Toulouse University. She received her PhD degree and HDR degree in automatic control in 1990 and 2000, respectively, from University Paul Sabatier, Toulouse. Her current research interests include constrained control and robust control of processes with limited information, with particular interest in applications on aeronautical systems, robotic, electronic, biochemical and environmental processes. She has been serving as member of the IFAC technical committees on "Biosystems and Bioprocesses" and on "Modelling and Control of Environmental Systems", respectively from 2002 and 2005 and of the IEEE CSS-CEB from 2013. She is currently AE for IET Control Theory and Applications and for the IFAC Journal NAHS (Nonlinear Analysis: Hybrid systems). She is co-author of a book on saturated systems and of more than 50 journal papers, both in control theory and in process engineering.

Sophie Tarbouriech received the PhD degree in Control Theory in 1991 and the HDR degree (Habilitation à Diriger des Recherches) in 1998 from University Paul Sabatier, Toulouse, France. Currently, she is full-time researcher (Directeur de Recherche) in LAAS-CNRS, Toulouse. Her main research interests include analysis and control of linear and nonlinear systems with constraints (limited information), hybrid dynamical systems. She is currently Associate Editor for IEEE Transactions on Automatic Control, IEEE Transactions on Control Systems Technology, Automatica and European Journal of Control. She is also in the Editorial Board of International Journal of Robust and Nonlinear Control. She is also co-Editor-in-Chief of the French journal JECS (Journal Européen des Systèmes Automatisés). Since January 2017, she is Senior Editor of the journal IEEE Control Systems Letters. Since 1999, she is Associate Editor at the Conference Editorial Board of the IEEE Control Systems Society. She is a member of the IFAC Technical Committees on Robust Control and Nonlinear Systems. She is also member of the IEEE Technical Committee on Hybrid Systems.

Jean-Marc Biannic graduated from SUPAERO Engineering School in 1992 and received the PhD degree in Robust Control Theory with the highest honors in 1996 from SUPAERO as well. He joined ONERA as a research scientist in 1997 and received the HDR degree (French habilitation as PhD supervisor) from Paul Sabatier’s University of Toulouse in 2010. Jean-Marc Biannic has supervised 6 PhD students. He is the author or co-author of 20 journal papers, beyond 50 conference papers, many book chapters, teaching documents, a tutorial book on multivariable control and Matlab toolboxes. He received in 2011 the second research distinction grade(MR2) from ONERA and the “ERE” distinction from ISAE (Aeronautics and Space Institute) thanks to which he is recognized as a full-professor in PhD committees. Jean-Marc Biannic has participated to several European projects and Garteur Groups (on PIO and nonlinear control). From 2012 to 2016, he has led a research project involving 10 research scientists for the development of the SMAC toolbox (w3.onera.fr/smac) for systems modeling, analysis and control.

Christophe Prieur was born in Essey-les-Nancy, France, in 1974. He graduated in Mathematics from the Ecole Normale Supérieure de Cachan, France in 2000. He received the Ph.D degree in 2001 in Applied Mathematics from the Université Paris-Sud, France, and the “Habilitation à Diriger des Recherches” (HDR degree) in 2009. From 2002 he was an associate researcher CNRS at the laboratory SATIE, Cachan, France, and at the LAAS, Toulouse, France (2004-2010). In 2010 he joined the Gipsa-lab, Grenoble, France where he is currently a senior researcher of the CNRS (since 2011). His current research interests include nonlinear control theory, hybrid systems, and control of partial differential equations. He was the Program Chair of the 9th IFAC Symposium on Nonlinear Control Systems (NOLCOS 2013) and of the 14th European Control Conference (ECC 2015). He has been a member of the IEEE-CSS Conference Editorial Board and an associate editor for IMA J. Mathematical Control and Information. He is currently a member of the EUCA-CEB, an associate editor for the IEEE Trans. on Automatic Control, European J. of Control, and IEEE Trans. on Control Systems Technology, and a senior editor for the IEEE Control Systems Letters.